# STABILITY OF BEAMS UNDER FORCE F IN MIDDLE OF SPAN COMBINED WITH END MOMENTS $M_a$ AND $M_b$

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**Abstract:** Calculation of the critical moments  $M_{cr}$  of the beams under various combinations of the force F in the middle of span and the end moments. Solution of the differential equations is presented in the graphical form. Two diagrams enable to obtain the factors  $C_1$  and  $C_2$ , which depend on the parameters  $\overline{M}$  and  $\psi$ . Approximate formula for the calculation of the critical moments depending on three factors  $C_1$ ,  $C_2$ ,  $C_3$  and three dimensionless parameters  $\kappa_{Wt}$ ,  $\zeta_g$ ,  $\zeta_j$  was derived in (2000). The formula together with the proposed values of factors  $C_1$ ,  $C_2$ ,  $C_3$  for basic loadings of beams with the monosymmetric I-sections were accepted for Eurocode (2007) and many national annexes. Presented results are valid for the beams with double symmetric I-sections. They may be used also for the continuous beams being slightly on the safe side. Numerical examples show that our results equal to the exact results of commercial computer programs. The procedure gives quickly practically exact results and therefore it is efficient tool for designers. This paper is twin of the paper (2022) where beams under uniform loading q and end moments are investigated.

Keywords: Stability of beam in bending, Critical moment, Uniform loading combined with end moments, Double symmetric I-section.

#### 1. Application of proposed procedure

## Numerical example 1:

Input values: beam with span L=6 m simply supported in vertical bending  $(k_y=1)$ , horizontal bending  $(k_z=1)$  and torsion  $(k_w=1)$  with rolled section IPE 200 is loaded by force F=15 kN acting on the surface of the upper flange  $(z_g=+100 \text{ mm})$ . The values of the end moments are  $M_a=-9.6$  kNm and  $M_b=0$  kNm (Fig.1). E=210 GPa, G=81 GPa, S 235,  $f_y=235$  MPa,  $\gamma_{M0}=1.0$ ,  $\gamma_{M1}=1.0$ . Cross-sectional properties of IPE 200 – DIN 1025, Part 5 (03/1994): h=200 mm, b=100 mm,  $t_f=8.5$  mm,  $t_w=5.6$  mm, buckling curve "a", cross-section Class 1,  $I_y=1$  943 cm<sup>4</sup>,  $W_{pl,y}=220.64$  cm<sup>3</sup>,  $M_{pl,y,Rk}=51.85$  kNm,  $I_z=142.4$  cm<sup>4</sup>,  $I_w=12.746$  cm<sup>6</sup>,  $I_t=6.846$  cm<sup>4</sup>. For double symmetric section parameter of section symmetry  $z_j=0$  mm, the factor  $C_3$  is not necessary.  $M_{Ed,max}=17.7$  kNm is in the middle of the span section x=3 m.

#### Numerical example 2:

Input values: the same as in the example 1, only difference is that point load of application of the uniform loading q is on the surface of the bottom flange of IPE 200 section ( $z_g = -100$  mm). The results of the example 2 are given below in the brackets. Dimensionless parameters of the beam:

$$\kappa_{wt} = \frac{\pi}{k_w L} \sqrt{\frac{EI_w}{GI_t}} = 0.364, \ \zeta_g = \frac{\pi z_g}{k_z L} \sqrt{\frac{EI_z}{GI_t}} = 0.385 \ (-0.385), \ \zeta_j = \frac{\pi z_j}{k_z L} \sqrt{\frac{EI_z}{GI_t}} = 0$$
 (1)

a) Calculation of the critical moment  $M_{cr}$  of the ideal beam:

Diagram parameters  $\overline{M}$  and  $\psi$ :  $|M_a = -9.6 \, \text{kNm}| > |M_b = 0 \, \text{kNm}| \rightarrow M = M_a = -9.6 \, \text{kNm}$ ,  $M_q = -9.6 \, \text{kNm} = -9.6 \, \text{kNm}$ . For parameters  $\psi = M_b / M_a = 0$ ,  $\overline{M} = M / (|M| + M_q) = -9.6 / (|-9.6| + 22.5) = -0.299$  we obtain the factors  $C_1 = 1.46$ ,  $C_2 = 0.78$  from Figs. 1, 2.

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$$\mu_{cr} = \frac{C_1}{k_z} \left[ \sqrt{1 + \kappa_{wt}^2 + \left( C_2 \xi_g + C_3 \zeta_j \right)^2} - \left( C_2 \xi_g + C_3 \zeta_j \right) \right] = 1,176, (2,052)$$
 (2)

Critical moment: 
$$M_{cr} = \mu_{cr} \frac{\pi \sqrt{EI_z GI_t}}{L} = 25,08 \text{ kNm}, (43,752 \text{ kNm})$$
 (3)

Critical moment calculated by LTBeam:  $M_{cr,LTBeam} = 24,938 \text{ kNm}, (43,391 \text{ kNm})$  (4)

b) Verification of the real beam:

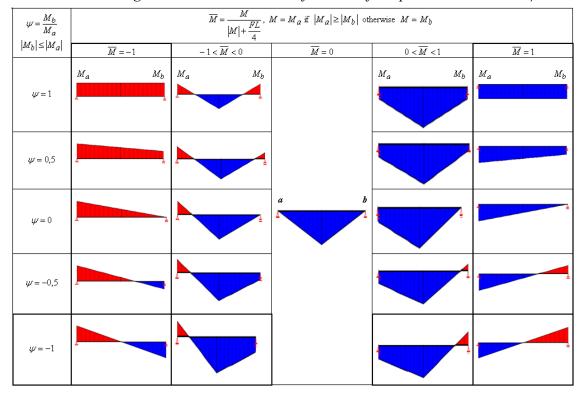
relative slenderness: 
$$\overline{\lambda}_{LT} = \sqrt{\frac{W_{pl,y}f_y}{M_{cr}}} = 1,438, (1,089), \text{ imperfection factor } \alpha_{LT} = 0,21,$$
 (5)

$$\Phi_{LT} = 0.5 \left[ 1 + \alpha_{LT} \left( \overline{\lambda}_{LT} - 0.2 \right) + \overline{\lambda}_{LT}^{2} \right] = 1,664, (1,186), \quad \chi_{LT} = \frac{1}{\Phi_{LT} + \sqrt{\Phi_{LT}^{2} - \overline{\lambda}_{LT}^{2}}} = 0.4, (0,604). \quad (6)$$

Design resistance of the beam: 
$$M_{b,Rd} = \chi_{LT} W_{pl,y} \frac{f_y}{\gamma_{M1}} = 20,735 \text{ kNm}, (31,308 \text{ kNm}).$$
 (7)

Utilization factor: 
$$U = \frac{M_{Ed}}{M_{h_{Rd}}} = \frac{17.7}{20.735} = 0.854 < 1.0, (0.565 < 1.0).$$
 (8)

*Tab.1: Bending moment distributions as function of the parameters*  $\overline{M}$  *and*  $\psi$ 



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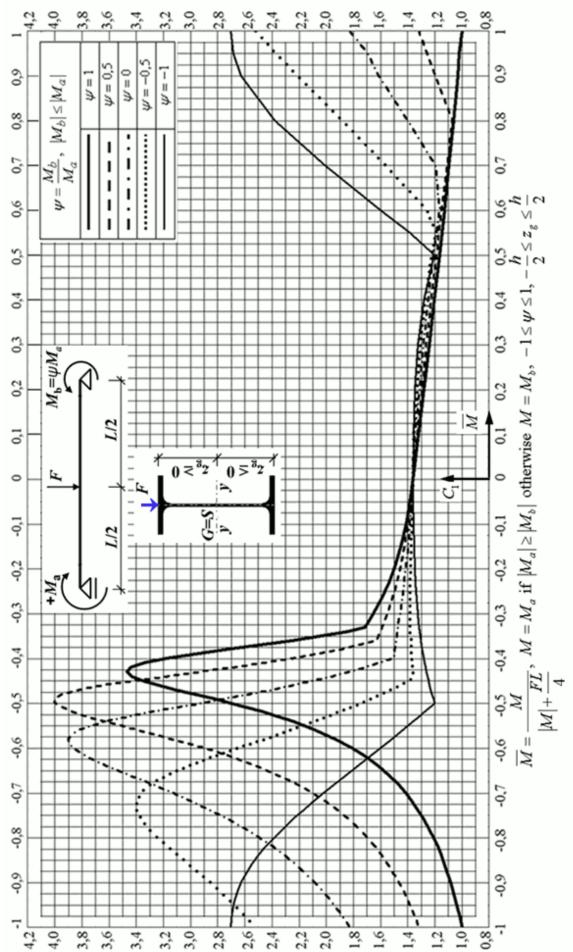
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EN 1999-1-1 (2007) and Amendment A1 (2009) and Amendment A2 (2013), Eurocode 9: Design of Aluminium Structures, Part 1.1: General Structural Rules, CEN Brussels.



₹ Fig.1: Factor C<sub>1</sub> for calculation of critical moment M<sub>er</sub> of simply supported beam with double symmetric I-section as function of parameters M and

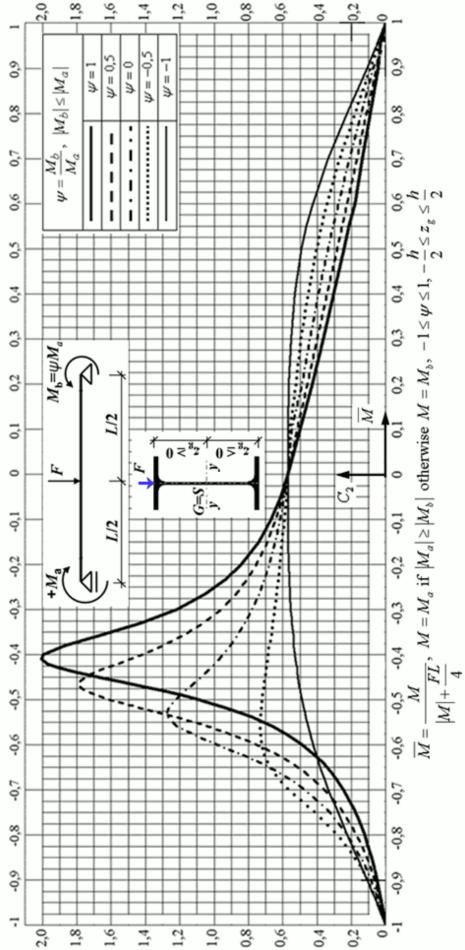


Fig.2: Factor  $C_2$  for calculation of critical moment  $M_{lpha}$  of simply supported beam with double symmetric I-section as function of parameters  $\overline{M}$  and  $\psi$