

INFLUENCE OF FLOW DIRECTION CHANGE ON THE PARTICLE SEGREGATION IN HORIZONTAL PIPES

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Abstract: In the pneumatic transport of solid particles, a gradual deposition of particles takes place. One of the factors affecting the intensity of particle segregations is the flow conditions upstream the pipeline. Numerical calculations allowed determining the intensity of particles deposition during transport in a horizontal pipeline with different elbow positioning at the inlet to the channel.

Keywords: pneumatic conveying, particle segregation, numerical calculations

1. Introduction

Horizontal pipelines are frequently occurring elements of the pneumatic transport system. During longer distance transport, a significant operational problem is the deposition of solid particles in the bottom of the pipeline (Borsuk et al., 2004). The most common side effects caused by the phenomenon of particle segregation, in addition to erosion and pulsation (Wydrych et al., 2017, Anweiler et al., 2004) may include a change in operations conditions in industrial installations and often their deterioration (Akili et al., 2001, Lain et al., 2012). The formation of particle rope is a problem, particularly with non-spherical particles (Kruggel-Emden et al., 2014). It was also compared both, experimental and numerical research of pulverized coal particles conveying in the horizontal channel (Akili et al., 2005). In the power industry, these problems can occur in the systems of dust-fired power boilers (Olcay et al., 2016). The aim of this work is a theoretical analysis of the phenomenon of gas-solid mixture flow in a horizontal pipe. In particular, the object of the research is to assess the impact of an elbow on the particle segregation process.

2. Mathematical model

The three-dimensional isothermal flow of the fixed gas-solid mixture is analyzed. The gas motion is described based on the conservation of mass and momentum (Borsuk et al., 2006). The basic system of equations is made up of the equation of flow continuity and the equation of momentum conservation. In this work, the system of motion is closed with the k- ϵ turbulence model (Borsuk et al., 2016).

The equation of continuity, motion and turbulence model can be presented in a generalized conservative form, isolating convective, diffusion and source elements

$$\frac{\partial}{\partial x}(\rho U\phi) + \frac{\partial}{\partial y}(\rho U\phi) + \frac{\partial}{\partial z}(\rho U\phi) = \frac{\partial}{\partial x}\left(\Gamma_{\phi}\frac{\partial\phi}{\partial x}\right) + \frac{\partial}{\partial y}\left(\Gamma_{\phi}\frac{\partial\phi}{\partial y}\right) + \frac{\partial}{\partial z}\left(\Gamma_{\phi}\frac{\partial\phi}{\partial z}\right) + S_{\phi} + S_{\phi \phi} \quad (1)$$

where ϕ is a generalized dependent variable, Γ_{ϕ} the diffusion transport coefficient, and the source element

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 S_{ϕ} captures all other – apart from convective and diffusion – components of differential equations. The coefficients Γ_{ϕ} and S_{ϕ} depend on the variable ϕ and determined for each of the equations according to table 1.

Tab. 1: Statement of equation's (1) coefficients.

Equation	¢	Γ_{ϕ}	S_{ϕ}	$S_{\phi p}$
Continuity	1	0	0	0
Momentum in the direction of axis x _i	ui	μ_{ef}	$F_{i} - \frac{\partial \overline{p}}{\partial x_{i}} + \frac{\partial}{\partial x_{j}} \left(\mu_{ef} \frac{\partial \overline{u_{j}}}{\partial x_{i}} \right)$	$\overline{S_{u_i,p}}$
Kinetic enegery of turbulence	k	$rac{\mu_{e\!f}}{\sigma_{\scriptscriptstyle k}}$	$\frac{\partial \overline{u}_i}{\partial x_j} \mu_t \left(\frac{\partial \overline{u}_i}{\partial x_j} + \frac{\partial \overline{u}_j}{\partial x_i} \right) - \rho \varepsilon$	0
Dissipation of the kinetic energy of turbulence	3	$rac{\mu_{e\!f}}{\sigma_{arepsilon}}$	$\frac{\varepsilon}{k} \left(C_1 \frac{\partial \overline{u}_i}{\partial x_j} \mu_t \left(\frac{\partial \overline{u}_i}{\partial x_j} + \frac{\partial \overline{u}_j}{\partial x_i} \right) - C_2 \rho \varepsilon \right)$	0

The particle motion is described by the Lagrange method. The equation of motion with considering the inertia force, aerodynamic drag force and gravity takes the form of (2)

$$\frac{du_p}{dt} = \frac{1}{\tau_p} \left(u - u_p \right) + g \tag{2}$$

where τ_p is the time of dynamic relaxation.

Interphase interactions were included in the form of additional source components $S_{\phi p}$ of the continuous phase equations. Fig. 1 presents the flow systems analyzed in the calculations. Both systems contain a straight horizontal pipe with the length of 60 diameters and an elbow located upstream to this pipe.



Fig. 1: Flow systems considered in calculations

3. Results of the research and their analysis

In order to analyze the phenomenon of particle segregation during pneumatic conveying in the gas stream, a series of numerical calculations were carried out using the ANSYS-Fluent program (Ansys, 2017). Spherical particles with the density 1550 kg/m³ were used. Fig. 2 presents particle concentration distribution for diameter $d = 90 \mu m$ in selected cross-sections of a flow system. Fig. 2a shows particle concentration distribution in a horizontal pipe, Fig. 2b a system with a vertical elbow and downflow, while Fig. 2c – a system with a vertical elbow and upflow.

In the case of the flow of particles through the horizontal pipe, it is seen that most of the particles move to the bottom of the pipe at the moment of moving away from the inlet. During conveying through the vertical elbow with downflow the areas of maximum concentration move to the bottom part of the horizontal channel while during upflow the areas of maximum concentration can be observed initially in the upper section of the channel and then, as the distance from the inlet increases, particles deposit gradually in the lower part of the channel.





In addition to the qualitative determination of segregation level, an additional procedure was used to determine two values: the maximum Y_{max} and the mean value Y_{av} of the particle concentration in the cross-section of the channel.

$$\phi = \frac{Y_{max}}{Y_{av}} \tag{3}$$

The ϕ value shows us the degree of unevenness in the distribution of the particle concentrations in particular cross-sections. Fig. 3 shows the changes in the coefficient of inhomogeneity of the concentration distribution along the horizontal pipeline during conveying for 40 μ m particles, while Fig. 4 shows the values of the relevant data for 90 μ m particles.



Fig. 3: . Concentration inhomogeneity coefficient in selected cross-sections horizontal pipe, $d = 40 \ \mu m$



Fig. 4: Concentration inhomogeneity coefficientin selected cross-sections horizontal pipe, $d = 90 \ \mu m$

Fig. 3 and Fig. 4 show a large increase of the concentration inhomogeneity coefficient in the initial section of the pipeline, which is the effect of the centrifugal force when changing the flow direction. For particles with a diameter 40 μ m (Fig. 3) the ϕ value decreases at a distance larger than 20 diameters from the inlet. The value of the coefficient ϕ practically does not depend on the direction of the inflow. The calculated ϕ values practically do not differ from the values corresponding to the flow in the straight horizontal pipe.

For particles with a diameter 90 μ m (Fig. 4) the effect of centrifugal force is more visible. The ϕ values in the initial section of the flow system are greater. The influence of gravity on the gradual particle deposition along the pipeline, in this case, is more important.

4. Conclusions

Numerical research carried out allows formulating the following conclusions. Change of the flow direction causes a significant increase of the particle concentration in the external side of the elbow. The effect of gradual particle segregation and its sedimentation at the bottom side of the pipeline depends on the particle size distribution. In the case of a system with a vertical elbow, the phenomenon of segregations depends on the flow direction at the inlet.

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